

OXFORD CAMBRIDGE AND RSA EXAMINATIONS

Advanced Subsidiary General Certificate of Education Advanced General Certificate of Education

MATHEMATICS

2640

Mechanics 4

Wednesday

21 JANUARY 2004

Afternoon

1 hour 20 minutes

Additional materials:
Answer booklet
Graph paper
List of Formulae (MF8)

TIME

1 hour 20 minutes

INSTRUCTIONS TO CANDIDATES

- Write your Name, Centre Number and Candidate Number in the spaces provided on the answer booklet.
- Answer all the questions.
- Give non-exact numerical answers correct to 3 significant figures unless a different degree of accuracy is specified in the question or is clearly appropriate.
- Where a numerical value for the acceleration due to gravity is needed, use 9.8 m s⁻².
- You are permitted to use a graphic calculator in this paper.

INFORMATION FOR CANDIDATES

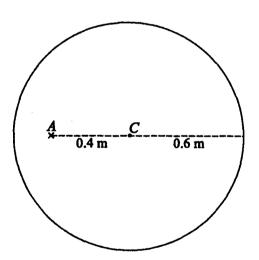
- The number of marks is given in brackets [] at the end of each question or part question.
- The total number of marks for this paper is 60.
- Questions carrying smaller numbers of marks are printed earlier in the paper, and questions carrying larger numbers of marks later in the paper.
- You are reminded of the need for clear presentation in your answers.

This question paper consists of 4 printed pages.

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- A wheel is rotating about a fixed axis, and is slowing down with constant angular deceleration $0.3 \,\mathrm{rad}\,\mathrm{s}^{-2}$.
 - (i) Find the angle the wheel turns through as its angular speed changes from 8 rad s⁻¹ to 5 rad s⁻¹.
 - (ii) Find the time taken for the wheel to make its final complete revolution before coming to rest.
 [3]
- A rod AB of variable density has length 2 m. At a distance x metres from A, the rod has mass per unit length (0.7 0.3x) kg m⁻¹. Find the distance of the centre of mass of the rod from A. [5]
- From a speedboat, a ship is sighted on a bearing of 045°. The ship has constant velocity 8 m s⁻¹ in the direction with bearing 120°. The speedboat travels in a straight line with constant speed 15 m s⁻¹ and intercepts the ship.
 - (i) Find the bearing of the course of the speedboat. [4]
 - (ii) Find the magnitude of the velocity of the ship relative to the speedboat. [3]
- The region between the curve $y = \frac{x^2}{a}$ and the x-axis for $0 \le x \le a$ is occupied by a uniform lamina with mass m. Show that the moment of inertia of this lamina about the x-axis is $\frac{1}{2}ma^2$. [7]

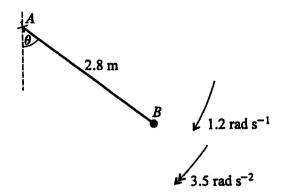
5



A uniform circular disc has mass 4 kg, radius 0.6 m and centre C. The disc can rotate in a vertical plane about a fixed horizontal axis which is perpendicular to the disc and which passes through the point A on the disc, where AC = 0.4 m. A frictional couple of constant moment 4.8 N m opposes the motion. The disc is released from rest with AC horizontal (see diagram).

- (i) Find the moment of inertia of the disc about the axis through A. [2]
- (ii) Find the angular acceleration of the disc immediately after it is released. [3]
- (iii) Find the angular speed of the disc when C is first vertically below A. [4]

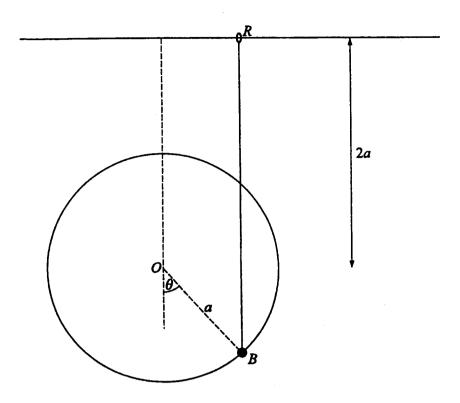
- A rigid body consists of a uniform rod AB, of mass 15 kg and length 2.8 m, with a particle of mass 5 kg attached at B. The body rotates without resistance in a vertical plane about a fixed horizontal axis through A.
 - (i) Find the distance of the centre of mass of the body from A. [2]
 - (ii) Find the moment of inertia of the body about the axis. [2]



At one instant, AB makes an acute angle θ with the downward vertical, the angular speed of the body is 1.2 rad s⁻¹ and the angular acceleration of the body is 3.5 rad s⁻² (see diagram).

- (iii) Show that $\sin \theta = 0.8$. [3]
- (iv) Find the components, parallel and perpendicular to BA, of the force acting on the body at A. [6]

[Question 7 is printed overleaf.]



A small bead B, of mass m, slides on a smooth circular hoop of radius a and centre O which is fixed in a vertical plane. A light elastic string has natural length 2a and modulus of elasticity mg; one end is attached to B, and the other end is attached to a light ring R which slides along a smooth horizontal wire. The wire is in the same vertical plane as the hoop, and at a distance 2a above O. The elastic string BR is always vertical, and OB makes an angle θ with the downward vertical (see diagram).

- (i) Show that $\theta = 0$ is a position of stable equilibrium. [7]
- (ii) Find the approximate period of small oscillations about the equilibrium position $\theta = 0$. [7]

 $0 \qquad \qquad \omega_{_{1}}{^{2}}=\omega_{_{0}}{^{2}}+2\dot{\omega}\theta$

$$u_1^2 = \omega_0^2 + 2\dot{\omega}\theta$$
 (equivalent of the linear $v^2 = u^2 + 2as$)

$$8^2 = 5^2 + 2 \times 0 \cdot 3 \times \theta$$

$$\theta = 65 \text{ rad}$$

For the final complete revolution ...

$$\theta = \omega_{\scriptscriptstyle 1} t - {\textstyle \frac{1}{2}} \, a t^2 \qquad \qquad (\text{"} \, s = v t - {\textstyle \frac{1}{2}} \, a t^2 \, \text{"})$$

$$2\pi = 0 - \frac{1}{2} \times 0 \cdot 3 \times t^2$$

$$t^2 = \frac{4\pi}{0.3}$$

$$t = 6 \cdot 4720... = 6 \cdot 47 \text{ s}$$

2 mass of rod = $\int_{0}^{2} 0 \cdot 7 - 0 \cdot 3x \, dx = \left[0 \cdot 7x - 0 \cdot 15x^{2} \right]_{0}^{2} = 0 \cdot 8$

$$0 \cdot 8\overline{x} = \int_{0}^{2} x \ 0 \cdot 7 - 0 \cdot 3x \ dx = 0 \cdot 35x^{2} - 0 \cdot 1x^{3} \Big|_{0}^{2} = 0 \cdot 6$$

$$\overline{x} = 0.75$$

[5]

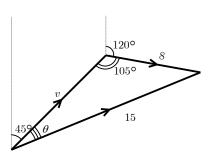
[2]

[3]

 ${\bf 3} \qquad \qquad {\rm speedboat} \, {\bf V}_{\rm ship} \, {\stackrel{\circ}{}} \, {\rm ship} \, {\bf V}_{\rm sea} \qquad {\rm speedboat} \, {\bf V}_{\rm sea}$

Sine Rule ...

$$\frac{\sin \theta}{8} \pi \frac{\sin 105}{15}$$
$$\theta \theta 31 \omega 008...$$



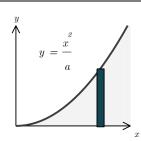
So the bearing of the speedboat = $45^{\circ} + \theta = 076 \cdot 008... = 076 \cdot 0^{\circ}$

[4]

$$\frac{v}{\sin 75} = \frac{15}{\sin 105}$$

$$v = 10 7858... = 10 8 \text{ ms}^{-1}$$
 [3]

4



area of lamina =
$$\frac{\theta}{1 + \frac{1}{100}} x^2 / a dx = \frac{1}{3} a^2$$

mass of 'elemental strip' =
$$\rho y \delta x = \frac{3m}{a^2} \cdot \frac{x^2}{a} \cdot \delta x$$

M.o.I =
$$\int_{0}^{a} \frac{4}{3} \frac{1}{2} y^{2} \cdot \frac{3mx^{2}}{a^{3}} dx = \frac{m}{a^{5}} \int_{0}^{a} x^{6} dx = \frac{1}{7} ma^{2}$$
 (show)

[7]

 $\mathbf{5}$

$$I_{C} = \frac{1}{2} mr^{2} = \frac{1}{2} \times 4 \times 0 \cdot 6^{2} = 0 \cdot 72$$

parallel axes theorem \dots

$$I_A = I_c + m \times 0 \cdot 4^2 = \mathbf{1} \cdot \mathbf{36} \text{ kg m}^2$$
 [2]

at the instant of release ...

$$\begin{aligned} C &= I_{\alpha} \\ 39 \cdot 2 \times 0 \cdot 4 &- 4 \cdot 8 = 1 \cdot 36 \times \alpha \\ & \alpha = 8 \cdot \mathbf{00} \ \mathbf{rad} \ \mathbf{s}^{\text{-1}} \end{aligned}$$

energy considerations ...

gain in K.E. = loss in G.P.E. - work done by friction

$$\frac{1}{2}I\omega^2 = 4 \times 9 \cdot 8 \times 0 \cdot 4 - 4 \cdot 8 \times \frac{\pi}{2}$$

$$\omega^2 = 11 \cdot 9708...$$

 $\omega = 3 \cdot 45989... = 3 \cdot 46 \text{ rad s}^{-1}$ [4]

 $M\,\overline{x} = \sum m_i$

$$M\,\overline{x} = \sum m_{\!\scriptscriptstyle i}\,x_{\!\scriptscriptstyle i} \qquad \qquad 20\,\overline{x} = 15\times 1\cdot 4 \ + \ 5\times 2\cdot 8$$

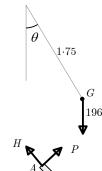
 $\bar{x}=1.75$

$$I = \frac{4}{3} m l^2 + 5 \ 2 l^{\ 2} = \frac{4}{3} \times 15 \times 1 \cdot 4^2 \ + \ 20 \times 1 \cdot 4^2 = \mathbf{78 \cdot 4} \ \ \mathbf{kg} \ \mathbf{m^2}$$

[2]

[2]

[3]



$$C = I_{\alpha}$$

 $196 \ 1 \cdot 75 \sin \theta \ = 78 \cdot 4 \times 3 \cdot 5$

$$\sin \theta = 0.8$$
 (show)

N2 radially BA

$$H - mg\cos\theta = m\omega^2 r$$

$$H = 20 \times 9 \cdot 8 \times 0 \cdot 6 + 20 \times 1 \cdot 2^2 \times 1 \cdot 75 = 168 \text{ N}$$

N2 normal

$$mg\sin\theta - P = mr\dot{\omega}$$

$$P = 20 \times 9 \cdot 8 \times 0 \cdot 8 - 20 \times 1 \cdot 75 \times 3 \cdot 5 = \mathbf{34} \cdot \mathbf{3}$$

[6]

[3]

Potential Energy Function (relative to position with $\theta = 0$)

$$V = mga \ 1 - \cos\theta + \frac{1}{2} \cdot \frac{mg}{2a} \cdot a \cos\theta^2 - a^2$$

$$= \frac{1}{4} mga \cos^2\theta - 4\cos\theta + 3$$

$$-\frac{\pi}{2} \le \theta \le \frac{\pi}{2}$$

$$\frac{\mathrm{d}V}{\mathrm{d}\theta} = \frac{1}{2} mga - \cos\theta \sin\theta + 2\sin\theta = \frac{1}{2} mga \sin\theta + 2\cos\theta$$

 $\frac{\mathrm{d}V}{\mathrm{d}\theta}\Big|_{\theta=0} = 0$ and therefore $\theta=0$ is an equilibrium position.

$$\frac{\mathrm{d}^2 V}{\mathrm{d}\theta^2} = \frac{1}{2} mga \left[2\cos\theta + \sin^2\theta - \cos^2\theta \right] \qquad \frac{\mathrm{d}^2 V}{\mathrm{d}\theta^2} \bigg|_{\theta=0} = \frac{1}{2} mga > 0$$

So the potential energy function has a local **minimum** at $\theta = 0$ and therfore we have a position of **stable equilibrium**.

conservation of mechanical energy ...

$$\frac{1}{4}mga \cos^2\theta - 4\cos\theta + 3 + \frac{1}{2}m a\dot{\theta}^2 = \text{constant}$$

differentiating ...

$$\frac{1}{2} mga \dot{\theta} 2 \sin \theta - \sin \theta \cos \theta + ma^2 \dot{\theta} \ddot{\theta} = 0$$

$$\ddot{\theta} = \frac{g}{2a} \sin \theta 2 - \cos \theta$$

for small oscillations $\sin \theta \approx \theta$ and $\cos \theta \approx 1$

and so
$$\ddot{ heta} pprox \left[rac{g}{2a}
ight] heta$$
 .

The motion is therefore approximately SHM with period $T = \frac{2\pi}{\sqrt{\frac{g}{2a}}} = 2\pi \sqrt{\frac{2a}{g}}$

[2]

[7]